

High-pressure control of an $\text{NH}_3/\text{H}_2\text{O}$ absorption heat transformer: effects on useful heat output and COP

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Objectives

This study experimentally investigates the influence of a proportional electronic expansion valve on the high-pressure level of an $\text{NH}_3/\text{H}_2\text{O}$ absorption heat transformer. In addition, the influence of the high-pressure level on both useful heat output and the coefficient of performance (COP) is analyzed.

Key data of absorption heat transformer

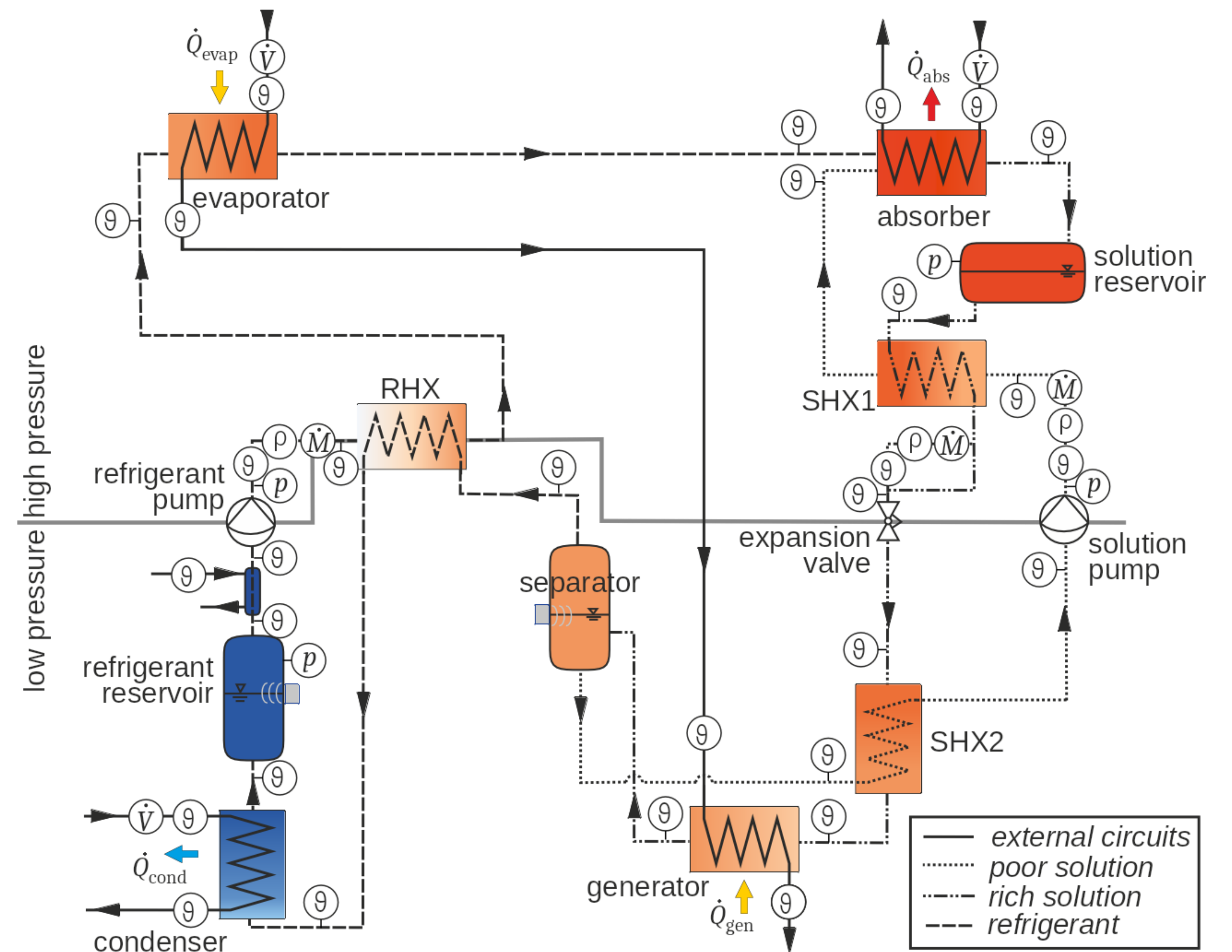
- Type: single-stage absorption heat transformer
- Working pair: $\text{NH}_3/\text{H}_2\text{O}$ (ammonia/water)
- Useful temperature level: $\leq 120^\circ\text{C}$
- Heat exchangers: plate heat exchangers (Alfa Nova HP27)
- Filling quantity: 9.8 kg with $\xi = 0.67$
- Expansion valve (EXV): electronic (Carel E2V24BSM00)
- Pumps: piston diaphragm pumps (Verder Hydra-Cell)

Performance parameters

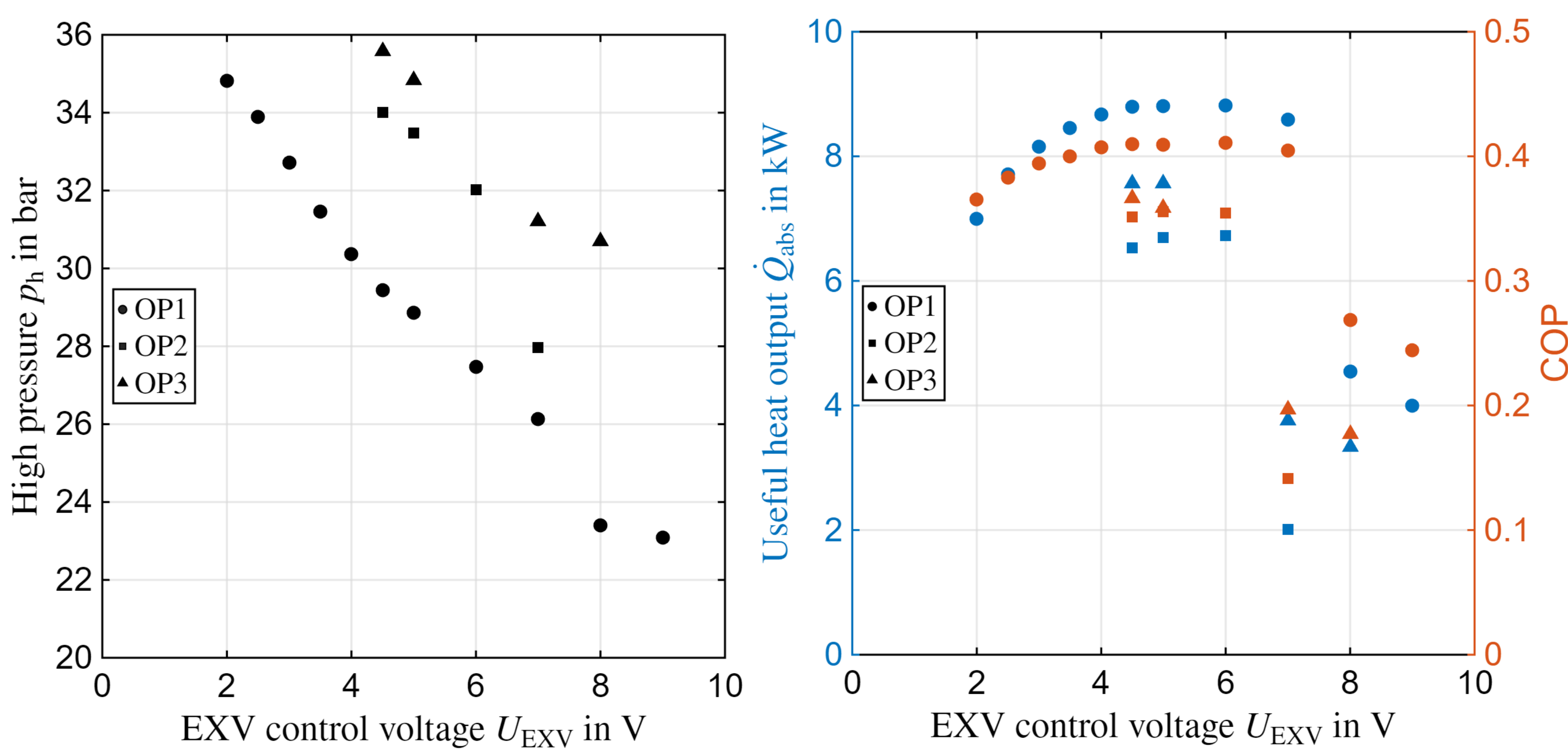
$$\dot{Q}_{\text{useful}} = \dot{Q}_{\text{abs}} \quad \text{COP} = \frac{|\dot{Q}_{\text{abs}}|}{|\dot{Q}_{\text{evap}}| + |\dot{Q}_{\text{gen}}|}$$

Operating points

OP	Recooling Condenser inlet	Driving heat Evaporator inlet	Useful heat Absorber outlet
OP1	$\vartheta_{\text{ext,cond,in}}: 25^\circ\text{C}$	$\vartheta_{\text{ext,evap,in}}: 80^\circ\text{C}$	$\vartheta_{\text{ext,abs,out}}: 100^\circ\text{C}$
OP2	$\vartheta_{\text{ext,cond,in}}: 25^\circ\text{C}$	$\vartheta_{\text{ext,evap,in}}: 80^\circ\text{C}$	$\vartheta_{\text{ext,abs,out}}: 110^\circ\text{C}$
OP3	$\vartheta_{\text{ext,cond,in}}: 25^\circ\text{C}$	$\vartheta_{\text{ext,evap,in}}: 90^\circ\text{C}$	$\vartheta_{\text{ext,abs,out}}: 120^\circ\text{C}$
OP1-3	$\dot{V}_{\text{ext,cond}} \approx 2000 \text{ l/h}$	$\dot{V}_{\text{ext,evap,gen}} \approx 2000 \text{ l/h}$	$\dot{V}_{\text{abs,ext}} \approx 1000 \text{ l/h}$



Results: influence of high-pressure level

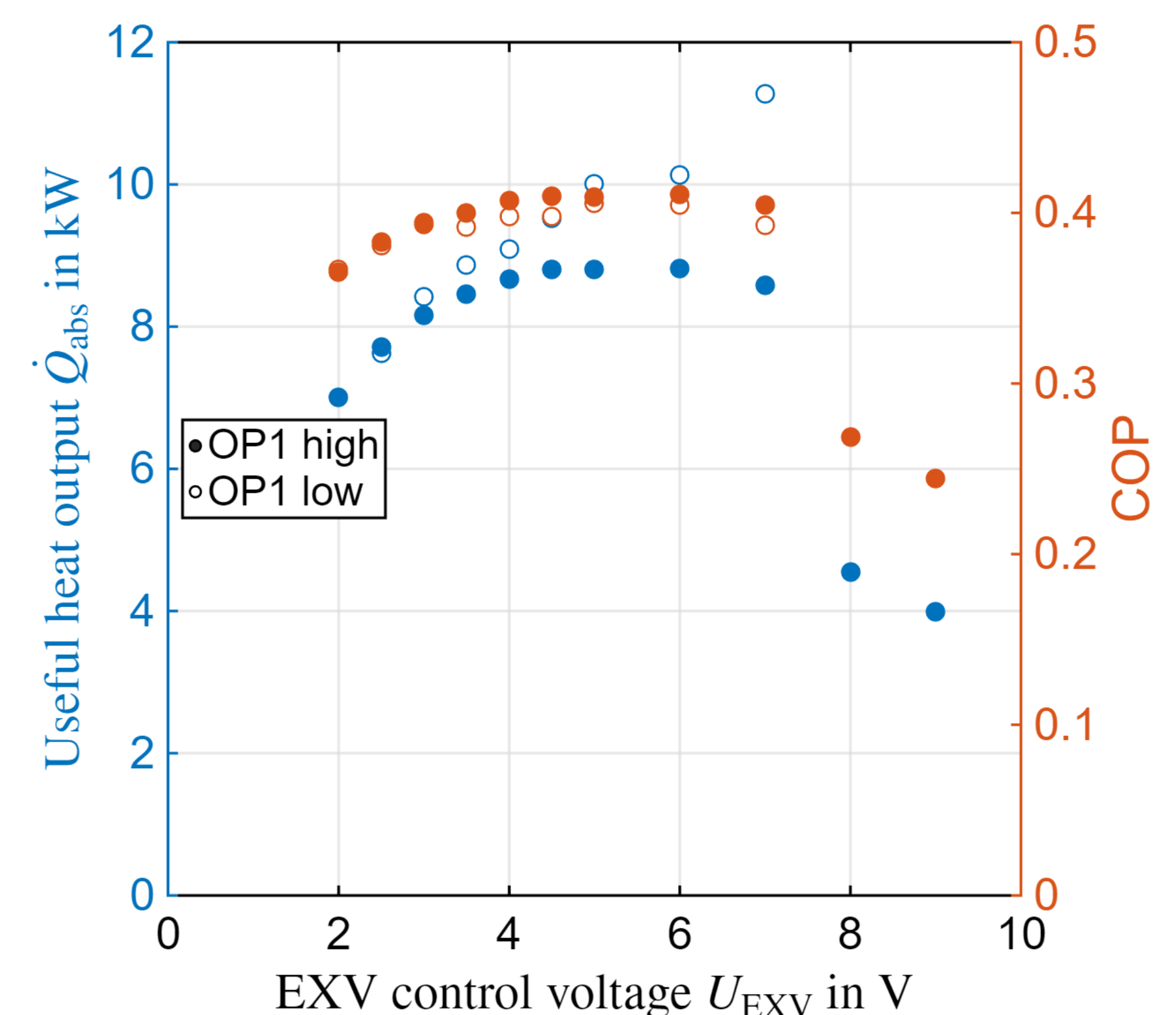
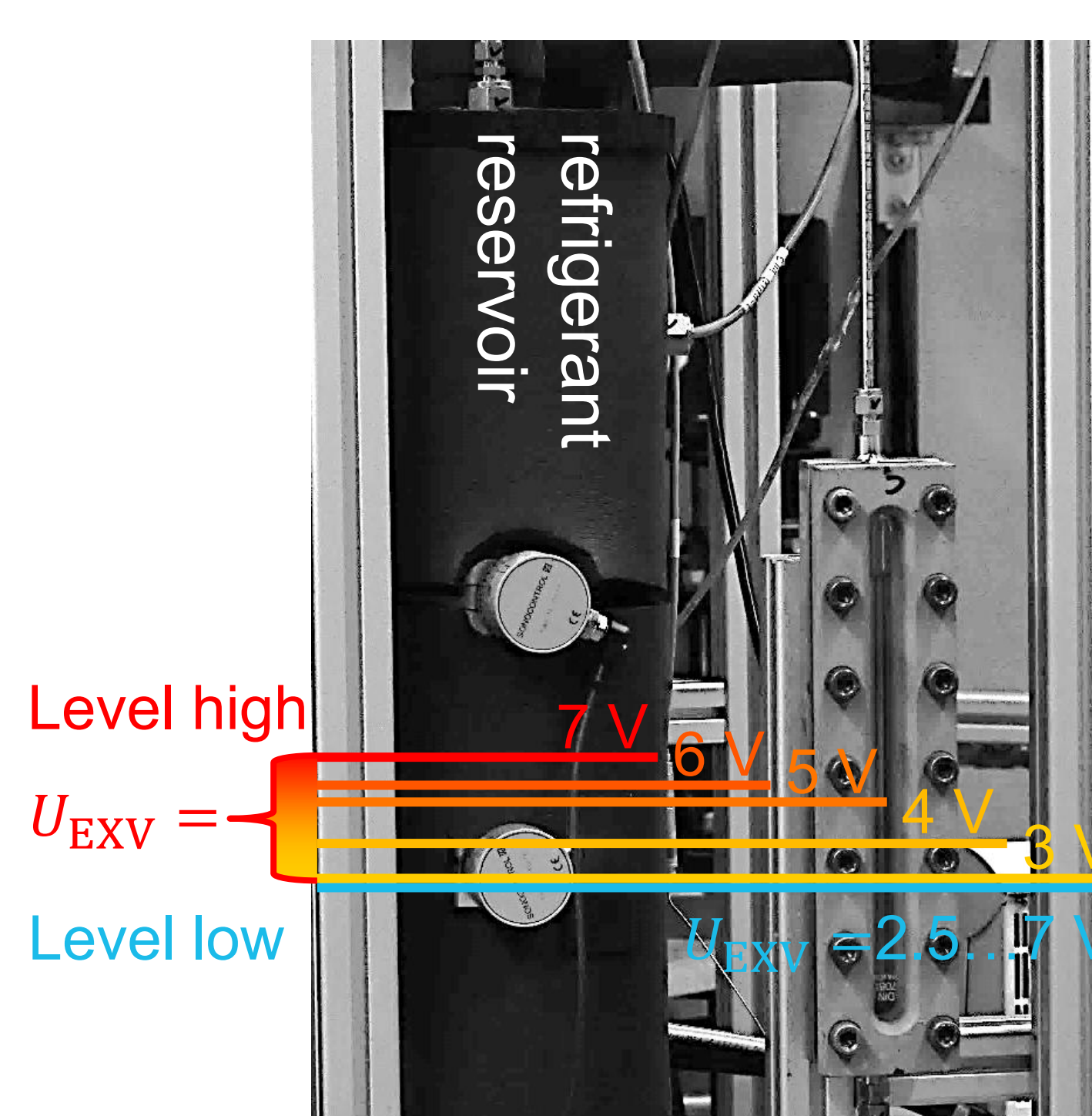


- High pressure p_h increases linearly with valve closure ($U_{\text{EXV}} \downarrow$) and increasing useful temperature level $\vartheta_{\text{ext,abs,out}} \uparrow$
- Mass flow $\dot{M}_{\text{rs}} \downarrow$, while $\dot{M}_{\text{ref}} = \text{const.}$ and $\dot{M}_{\text{ps}} \downarrow \rightarrow$ concentration $\xi_{\text{rs}} \uparrow \rightarrow$ high pressure $p_h \uparrow$
- System performance is governed by a trade-off between incomplete absorption and evaporation:
 - For $U_{\text{EXV}} > 6 \text{ V}$ and thus $p_h < p_{h,\text{opt}}$ incomplete absorption ($x_{\text{abs,out}} > 0.09 \text{ kg/kg}$)
 - For $U_{\text{EXV}} < 6 \text{ V}$ and thus $p_h > p_{h,\text{opt}}$ disproportionate decrease of vapor quality $x_{\text{evap,out}} \rightarrow$ less refrigerant vapor available for absorption
- \rightarrow This trade-off produces a characteristic curve with abrupt increase, broad plateau and subsequent drop-off for both useful heat output and COP
- \rightarrow Maximum for useful heat output and COP occur at medium valve opening independent of operating conditions for $\dot{M}_{\text{ref}} = \text{const.}$

Results: influence of filling level in the refrigerant reservoir

- Switching from filling level high to low increases the circulating refrigerant mass \rightarrow concentration $\xi_{\text{rs}} \uparrow \rightarrow$ high pressure $p_h \uparrow$
- Sensor position of refrigerant reservoir influences refrigerant mass flow \dot{M}_{ref} as refrigerant pump controls for constant filling level
 - For level high: $n_{\text{pump,ref}} = n_{\text{min}} \rightarrow \dot{M}_{\text{ref}} = \text{const.}$
 - For level low: $n_{\text{pump,ref}} \uparrow \rightarrow \dot{M}_{\text{ref}} \uparrow$
- \rightarrow For level low the disproportionate increase in refrigerant mass flow \dot{M}_{ref} dominates the trade-off between incomplete absorption and evaporation such that useful heat output increases when the valve opens
- \rightarrow Maximum for useful heat output occurs at maximum valve opening for $\dot{M}_{\text{ref}} \neq \text{const.}$, while COP still occurs at medium valve opening

Filling level



Conclusions

- \rightarrow The expansion valve controls the high pressure linearly and is an effective control variable
- \rightarrow To maximize useful heat output, a medium valve opening for $\dot{M}_{\text{ref}} = \text{const.}$ and maximum valve opening for $\dot{M}_{\text{ref}} \neq \text{const.}$ is desirable
- \rightarrow To maximize COP a medium valve opening is sufficient under all operating conditions